



Butterworth Pattern-based Simultaneous Damping and Tracking Controller Designs for Nanopositioning Systems

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The Butterworth filter is known to have maximally flat response. Incidentally, the same response is desired in precise positioning systems. This paper presents a method for obtaining a closed-loop Butterworth filter pattern using common control schemes for positioning applications, i.e., Integral Resonant Control (IRC), Integral Force Feedback (IFF), Positive Position Feedback (PPF), and Positive Velocity and Position Feedback (PVPF). Simulations show a significant increase in bandwidth over traditional design methods and verify the desired pole placement is achieved. The simulations also show a significant limitation of the achievable bandwidth in the case of IRC, IFF, and PPF. For this reason, only PVPF is considered in experimental analysis. Experiments are performed using a two-axis serial kinematic nanopositioning stage. The results show a significant improvement in bandwidth, from 123 to 370 Hz, and increased positioning accuracy, specifically at the turn-around point, in comparison to the sequentially designed control scheme.

Keywords: nanopositioning, damping control, tracking control, integral resonant control, positive position feedback, positive velocity and position feedback

1. INTRODUCTION

Lightly damped resonant systems are highly susceptible to excitation of the resonant modes. This behavior can severely limit performance and lead to catastrophic failure of the system. Various control techniques have been proposed in the literature to improve the performance of these systems. Open-loop control schemes, such as filtering and model-based inversion techniques (Croft et al., 2001), have been used. The performance of open-loop systems is limited by the sensitivity to perturbations in the system dynamics and rely on a having a sufficiently accurate model. Closed-loop controllers are more commonly used due to superior robustness and excellent stability (Preumont, 1997). In this work, we consider a piezoactuated serial kinematic nanopositioning platform as an example system. Nanopositioners exhibit highly resonant behavior due to a lightly damped dominant resonant mode at relatively low frequency and non-linear behavior inherent to the piezoelectric actuators.

The effect of the low-frequency resonant mode on the performance of a nanopositioner is that the scanning speed at which accurate tracking can be achieved is limited to approximately one-hundredth of the resonant frequency in open-loop operation (Moheimani, 2008). In applications which require high scanning speeds (Ando et al., 2002; Tuma et al., 2012), a controller which damps

the resonant mode is required. Numerous damping controllers have been presented in the literature. Integral Force Feedback (IFF) (Preumont et al., 2008) and Integral Resonant Control (IRC) (Aphale et al., 2007) are simple, easy to design control schemes which provide guaranteed stability and excellent robustness. Positive Position Feedback (PPF) (Fanson and Caughey, 1990) and Positive Velocity and Position Feedback (PVPF) (Bhikkaji et al., 2007) use the feedback of position, and position and velocity, respectively, to achieve closed-loop pole placements, which increase damping. This work will focus on IFF, IRC, PPF, and PVPF due to the simple design procedures. However, numerous other damping controllers exist and have been extensively reported, including Shunt control (Hagood and von Flotow, 1991), Resonant control (Aphale et al., 2008a), and H_∞ control (Sebastian and Salapaka, 2005). These damping controllers are not limited in use to nanopositioning. Successful implementation has been reported in applications, such as robotic manipulators (Pereira et al., 2011), disk drives (Numasato and Tomizuka, 2003), aircraft wings (Friedmann and Millott, 1995), scanning probe microscopes (Fleming et al., 2010), MEMS/NEMS (Ferreira and Aphale, 2011), and high-density memory storage devices (Sebastian et al., 2008).

Non-linear behavior, such as hysteresis and creep, is observed in nanopositioning due to the use of piezoelectric actuators (Figure 1). The positioning error caused by non-linearities is significant and must be considered when designing an appropriate controller. A number of methods have been proposed in the literature to reduce the effects of non-linear behavior on the positioning error. The hysteretic behavior of piezoactuators can be linearized by driving with charge actuation (Newcomb and Flinn, 1982; Fleming, 2013), rather than the more commonly used voltage actuation. It has also been shown that the augmentation of linear controllers with non-linear damping reduces the effects of non-linearities on positioning error (Kanestrom and Egeland, 1994; Vagia et al., 2013). Fuzzy control has been applied to nanopositioning, incorporating non-linearities in the modeling of the plant, to design controllers, which directly target the non-linear behavior (Yu et al., 2009). The most common method of reducing the effects of non-linear behavior in nanopositioning

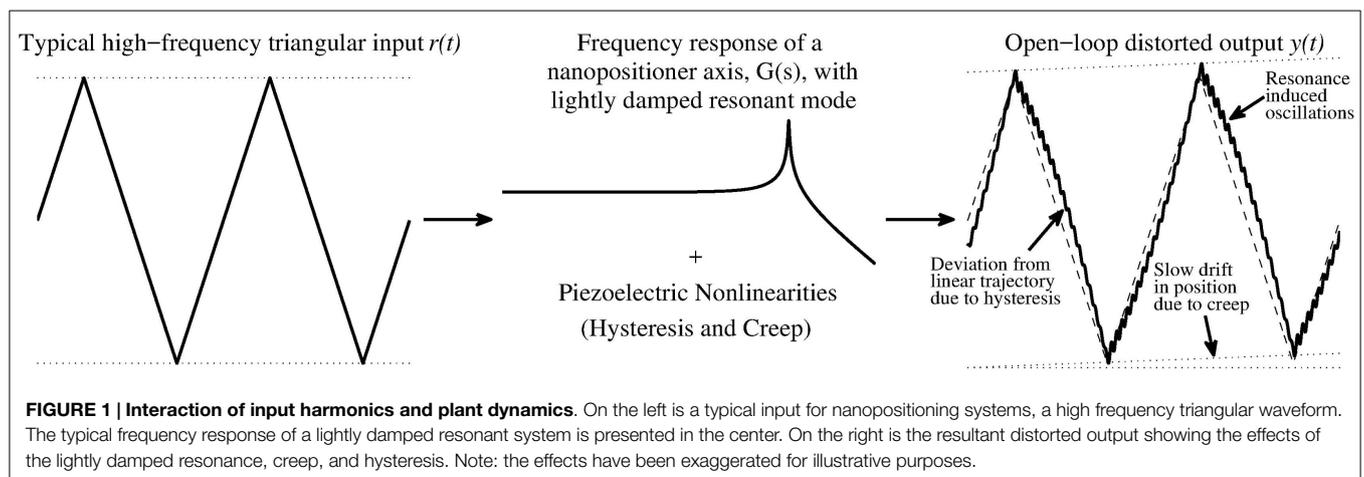
is through the use of reference tracking (Devasia et al., 2007). Debate is ongoing with regard to whether linear controllers are suitable for the control of non-linearities. However, numerous works have shown, experimentally, that integral tracking control reduces the effects of hysteresis on positioning error (Bhikkaji et al., 2007; Aphale et al., 2008a; Fleming, 2010).

Eielsen et al. (2014) showed that using Model Reference Control to obtain a closed-loop Butterworth filter pattern provides increased performance over traditional damping and tracking controllers, in terms of bandwidth and positioning error. The aim of this work is to provide a quick and simple method of deriving controllers to obtain a closed-loop Butterworth filter pattern. This allows the user to improve performance with limited knowledge of control theory.

One reason for the relatively poor performance of traditional controller designs in precise positioning is that the damping and tracking controllers are designed sequentially. This is due to the fact that many of the common damping controllers used in nanopositioning were either originally employed as standalone damping controllers in other applications or based on such controllers. In this case, the original design process of the damping controller is followed. When the tracking controller is added and the gain tuned, the damped characteristics of the closed-loop system are altered.

The performance of the system can be improved by simultaneously design the damping and tracking controllers. By utilizing a simultaneous design process, the desired damping and tracking characteristics can be achieved in closed-loop operation.

The paper is structured as follows: Section 2 provides an overview of the IRC, IFF, PPF, and PVPF control schemes. Methods are provided for each control scheme to obtain the closed-loop Butterworth filter pattern. Section 3 details the experimental nanopositioning setup. In Section 4, controller parameters are derived for both traditional and Butterworth-based control schemes. Simulations are provided to evaluate the performance of the proposed controller in comparison with the traditional design. Section 5 presents experimental results for the PVPF case, showing the effectiveness of the Butterworth-based controller design in implementation. Section 6 concludes the paper.



2. OBTAINING BUTTERWORTH PATTERN

A number of methods have been proposed for the control of second-order systems. These range from PI control, commonly used in commercial Atomic Force Microscopes, to optimal control methods, such as H_∞ (Sebastian and Salapaka, 2005). The ideal frequency response for tracking applications consists of unity gain at low frequencies to preserve the harmonics of the input, followed by a roll-off at high frequencies to reduce the effects of noise and high frequency unmodeled dynamics. The Butterworth filter is known to fit these criteria. The Butterworth filter is defined as having transfer function, $H(s)$, such that the poles of $H(s)H(-s)$ are equally spaced on a circle, with radius equal to the natural frequency of the Butterworth filter, in the Cartesian plane. In this section, a brief overview is given of the four control schemes used in this work and a method to achieve a closed-loop Butterworth filter pattern is presented.

First, we must consider the modeling of the plant. Nanopositioning platforms utilize collocated sensor/actuator pairs, which are modeled as the sum of multiple second-order transfer functions. However, the performance of a nanopositioner is limited by a dominant, low-frequency resonant mode. The nanopositioner can, therefore, be modeled as a single second-order system. This approach has been used extensively in the literature. The transfer function of the model, measured from the input voltage to the output displacement, is given by

$$G(s) = \frac{\sigma^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} \quad (1)$$

where ζ is the damping ratio, ω_n is the natural frequency, and σ is a constant gain determined by the physical characteristics of the plant. Although the controller design process, presented in the remainder of this section, utilizes a simplified model of the plant, the closed-loop performance is not degraded by the inclusion of higher-order modes.

2.1. Integral Resonant Control

Integral Resonant Control is an adaptation of IFF, whereby position feedback is used as opposed force feedback. In using position feedback, a feedthrough term is added to emulate the response of the force measurement. As this is no longer an inherent system parameter, the feedthrough can be arbitrarily chosen, giving greater control over the response. IRC has found wide use in a number of applications including nanopositioning (Aphale et al., 2008b; Bhikkaji and Moheimani, 2008), smart structures (Aphale et al., 2007), and flexible link manipulators (Pereira et al., 2011)

due to the simplicity of the design method and the stability and robustness properties. The structure of the IRC scheme is shown in **Figure 2A**, where $G(s)$ is the plant, d is an arbitrarily chosen constant gain, and $C_d(s)$ is an integral controller given by

$$C_{dIRC}(s) = \frac{k}{s}, \quad (2)$$

and the tracking controller, $C_t(s)$, is an integral controller given by

$$C_t(s) = \frac{k_t}{s}. \quad (3)$$

The traditional design procedure for IRC is as follows; first, the feedthrough, d , is chosen to be sufficiently large and negative. Then, the damping controller gain which provides maximum damping is found using the root-locus method. The tracking controller gain is found via trial-and-error. Further analysis has provided expressions for the feedthrough and damping controller gain, in terms of known parameters, which the maximum achievable damping (Namavar et al., 2014). Stability criteria are also provided which limits the range of tracking controller gains. However, it is known that the addition of the tracking controller affects the damped poles of the system, and it is observed that within the range of stability, the tracking controller gain and closed-loop damping ratio are inversely related. It is, therefore, necessary to design both the damping and tracking controllers simultaneously. In the remainder of this section, expressions are derived for the three tunable parameters, which will give a closed-loop Butterworth filter pattern.

From **Figure 2A** and the structure of the controllers as given previously, the closed-loop transfer function is derived as

$$G_{cl}^{num}(s) = k_t k \sigma^2$$

$$G_{cl}^{den}(s) = s^4 + (2\zeta\omega_n - dk)s^3 + (\omega_n^2 - 2\zeta\omega_n dk)s^2 + k(-d\omega_n^2 - \sigma^2)s + k_t k \sigma^2 \quad (4)$$

where G_{cl}^{num} denotes the numerator and G_{cl}^{den} the denominator of the closed-loop transfer function.

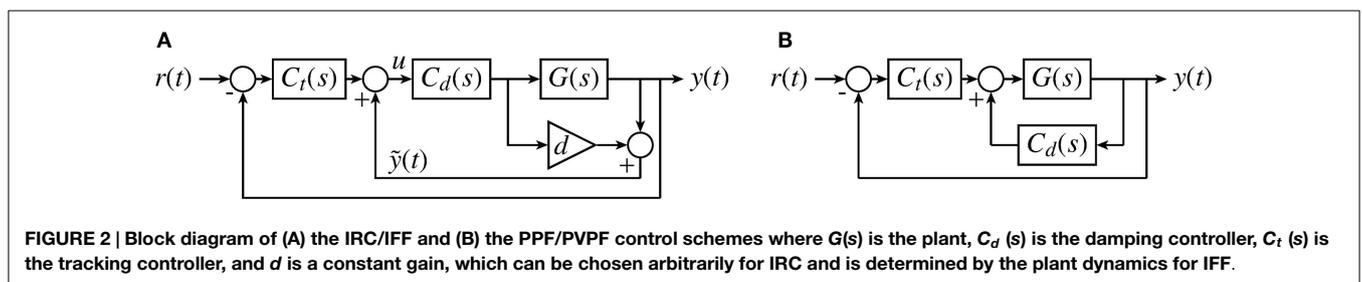
A fourth-order Butterworth filter has two sets of complex poles with the same natural frequency and differing damping ratios. The denominator of the transfer function is given by

$$P(s) = (s^2 + 2\gamma_1\omega_d s + \omega_d^2)(s^2 + 2\gamma_2\omega_d s + \omega_d^2) \quad (5)$$

$$= s^4 + \alpha\omega_d s^3 + \beta\omega_d^2 s^2 + \alpha\omega_d^3 s + \omega_d^4 \quad (6)$$

where

$$\alpha = 2\gamma_1 + 2\gamma_2, \beta = 2 + 4\gamma_1\gamma_2. \quad (7)$$



Equating the coefficients of the denominator of the closed-loop transfer function, in equation (4), and the characteristic equation of the Butterworth filter, in equation (6), gives the conditions for achieving a closed-loop Butterworth filter pattern, i.e.,

$$\alpha\omega_d = 2\zeta\omega_n - dk \tag{8}$$

$$\beta\omega_d^2 = \omega_n^2 - 2\zeta\omega_n dk \tag{9}$$

$$\alpha\omega_d^3 = k(-d\omega_n^2 - \sigma^2) \tag{10}$$

$$\omega_d^4 = k_t k \sigma^2. \tag{11}$$

Rearranging equations (8)–(11) to find the controller parameters gives

$$dk = 2\zeta\omega_n - \alpha\omega_d \tag{12}$$

$$dk = \frac{\omega_n^2 - \beta\omega_d^2}{2\zeta\omega_n} \tag{13}$$

$$dk = \frac{-k\sigma^2 - \alpha\omega_d^3}{\omega_n^2} \tag{14}$$

$$k_t = \frac{\omega_d^4}{k\sigma^2}. \tag{15}$$

Equating equations (12) and (13) and rearranging for ω_d gives

$$\beta\omega_d^2 - 2\zeta\omega_n\alpha\omega_d + 4\zeta^2\omega_n^2 - \omega_n^2 = 0 \tag{16}$$

The roots of equation (16) give the achievable damped natural frequency and are calculated as

$$\omega_d = \frac{2\zeta\omega_n\alpha \pm \sqrt{4\zeta^2\omega_n^2\alpha^2 - 4\beta(4\zeta^2\omega_n^2 - \omega_n^2)}}{2\beta}. \tag{17}$$

With the damped natural frequency, ω_d , now known, the damping controller gain, k , can be found by equating equations (12) and (14) giving

$$k = \frac{\alpha\omega_d\omega_n^2 - 2\zeta\omega_n^3 - \alpha\omega_d^3}{\sigma^2}. \tag{18}$$

The feedthrough term, d , is then calculated from equation (12) as

$$d = \frac{2\zeta\omega_n - \alpha\omega_d}{k}, \tag{19}$$

and the tracking controller gain, k_t , given by equation (15).

2.2. Integral Force Feedback

Integral Force Feedback was first proposed by Preumont et al. to damp the vibrations in space structures (Preumont et al., 1992). This method has since been refined (Preumont et al., 2008) and applied to other resonant systems, such as nanopositioning stages (Fleming, 2010). IFF with tracking control has the same structure as IRC, see **Figure 2A**, where C_d and C_t are both integral controllers. There are two main differences between IFF and IRC: first, IFF uses force feedback in the damping loop, whereas IRC uses position feedback. Second, the feedthrough term, d , is fixed and determined by properties of the system in IFF, IRC allows an arbitrary choice of feedthrough. In recent work, Optimal Integral Force Feedback (OIFF) has been developed (Teo et al., 2014). This

entails the addition of an arbitrarily chosen feedthrough term, β , such that the feedback in the damping loop, \tilde{y} , is given by

$$\tilde{y}(t) = (G + d_t)u(t) \tag{20}$$

where d_t is the total feedthrough gain, defined as $d_t = d + \beta$. The benefit of this method is an increase in the achievable damping and improved robustness. To achieve a closed-loop Butterworth pattern, the OIFF control scheme can be designed using the same method as IRC. The feedthrough, d_t , is calculated using equation (19). The additional feedthrough is then $\beta = d_t - d$. The integral damping and tracking controllers are unchanged.

2.3. Positive Position Feedback

Positive Position Feedback was originally developed for the control of large space structures (Goh and Caughey, 1985; Fanson and Caughey, 1990). It has since been successfully applied to nanopositioning stages (Aphale et al., 2008a). Traditionally, PPF uses a pole placement algorithm to design the damping controller, C_{dppf} . The closed-loop pole placement can be chosen arbitrarily. The controller is derived by equating the coefficients of the desired characteristic equation with those of the denominator of the closed-loop transfer function and solving for the controller parameters. However, due to the limitations of the PPF controller, the desired pole placement cannot always be achieved. The tracking controller gain is found via trial-and-error.

To obtain a closed-loop Butterworth filter pattern, the same method as for IRC is used, i.e., the denominator of the closed-loop transfer function is equated to the characteristic equation of the Butterworth filter. However, due to additional complexity of the higher-order system, some parameters must be found numerically.

The structure of the PPF control scheme with tracking is shown in **Figure 2B** where the PPF controller, C_d has transfer function

$$C_{dppf}(s) = \frac{\Gamma_1}{s^2 + 2\zeta_c\omega_c s + \omega_c^2}, \tag{21}$$

and the tracking controller, C_t , is an integrator given by

$$C_t(s) = \frac{k_t}{s}. \tag{22}$$

The closed-loop transfer function is given by

$$\begin{aligned} G_{cl}^{num}(s) &= k_t\sigma^2(s^2 + 2\zeta_c\omega_c s + \omega_c^2) \\ G_{cl}^{den}(s) &= s^5 + (2\zeta\omega_n + 2\zeta_c\omega_c)s^4 \\ &\quad + (2\zeta\omega_n 2\zeta_c\omega_c + \omega_n^2 + \omega_c^2)s^3 \\ &\quad + (2\zeta\omega_n\omega_c^2 + 2\zeta_c\omega_c\omega_n^2 + k_t\sigma^2)s^2 \\ &\quad + (\omega_n^2\omega_c^2 - \sigma^2\Gamma_1 + k_t\sigma^2 2\zeta_c\omega_c)s \\ &\quad + k_t\sigma^2\omega_c^2 \end{aligned} \tag{23}$$

The fifth-order Butterworth filter has the following characteristic equation

$$\begin{aligned} P(s) &= (s + \omega_d)(s^2 + 2\phi_1\omega_d s + \omega_d^2)(s^2 + 2\phi_2\omega_d s + \omega_d^2) \\ &= s^5 + (\alpha + 1)\omega_d s^4 + (\alpha + \beta)\omega_d^2 s^3 \\ &\quad + (\alpha + \beta)\omega_d^3 s^2 + (\alpha + 1)\omega_d^4 s + \omega_d^5 \end{aligned} \tag{24}$$

where

$$\alpha = 2\phi_1 + 2\phi_2 \quad (25)$$

$$\beta = 2 + 4\phi_1\phi_2. \quad (26)$$

Substituting the values for ϕ_1 , ϕ_2 , and ω_d gives the characteristic equation in the form

$$P(s) = s^5 + K_4s^4 + K_3s^3 + K_2s^2 + K_1s + K_0. \quad (27)$$

Equating the denominator coefficients of equation (23) with equation (27) and rearranging for the controller parameters gives

$$\begin{aligned} 2\zeta_c\omega_c &= K_4 - 2\zeta\omega_n \\ \omega_c^2 &= K_3 - \omega_n^2 - 2\zeta\omega_n 2\zeta_c\omega_c \\ k_t &= K_0/\sigma^2\omega_c^2 \\ \Gamma_1 &= (\omega_n^2\omega_c^2 + 2\zeta_c\omega_c\sigma^2k_t - K_1)/\sigma^2 \end{aligned} \quad (28)$$

As PPF does not allow arbitrary pole placement, a closed-loop Butterworth filter pattern can only be achieved for a specific damped natural frequency, ω_d . This is found numerically such that

$$|S_i| = \omega_d, \quad i = 1, \dots, 5 \quad (29)$$

where s_i are the roots of denominator of equation (23).

2.4. Positive Velocity and Position Feedback

Positive Velocity and Position Feedback (PVPF) is an adaptation of PPF. The addition of velocity feedback allows truly arbitrary pole placement to be achieved, where there is a limitation of the achievable pole placement of PPF. Successful implementation of the PVPF control scheme in nan positioning applications has been widely reported in the literature (Bhikkaji et al., 2007; Aphale et al., 2008a; Bazaeei et al., 2012). The design procedure of the PVPF controller is similar to that of the PPF controller. The desired closed-loop pole placement is chosen, typically by reducing the real component of the plant poles by a sufficiently large amount, such that the damping ratio is increased while maintaining the damped natural frequency of the plant. The tracking controller is designed via trial-and-error.

As PVPF allows arbitrary pole placement, the closed-loop Butterworth filter pattern can be easily achieved using a method similar to the traditional design. The only difference is that the tracking controller must be taken into account in the design process.

The structure of the PVPF control scheme with tracking is shown in **Figure 2B**, where the damping controller, C_d has transfer function

$$C_{d_{PVPF}}(s) = \frac{\Gamma_2s + \Gamma_1}{s^2 + 2\gamma\omega_c s + \omega_c^2}, \quad (30)$$

and the tracking controller is an integrator given by

$$C_t(s) = \frac{k_t}{s}. \quad (31)$$

The closed-loop transfer function is

$$\begin{aligned} G_{cl}^{num}(s) &= k_t\sigma^2(s^2 + 2\gamma\omega_c s + \omega_c^2) \\ G_{cl}^{den}(s) &= s^5 + (2\zeta\omega_n + 2\gamma\omega_c)s^4 + (2\zeta\omega_n 2\gamma\omega_c + \omega_n^2 + \omega_c^2)s^3 \\ &\quad + (2\zeta\omega_n\omega_c^2 + 2\gamma\omega_c\omega_n^2 + k_t\sigma^2)s^2 \\ &\quad + (\omega_n^2\omega_c^2 - \sigma^2\Gamma_1 + k_t\sigma^2 2\gamma\omega_c)s + k_t\sigma^2\omega_c^2 \end{aligned} \quad (32)$$

The fifth-order Butterworth filter has the following characteristic equation

$$P(s) = (s + \omega_d)(s^2 + 2\phi_1\omega_d s + \omega_d^2)(s^2 + 2\phi_2\omega_d s + \omega_d^2) \quad (33)$$

$$\begin{aligned} &= s^5 + (\alpha + 1)\omega_d s^4 + (\alpha + \beta)\omega_d^2 s^3 \\ &\quad + (\alpha + \beta)\omega_d^3 s^2 + (\alpha + 1)\omega_d^4 s + \omega_d^5 \end{aligned} \quad (34)$$

where α and β are given by equations (25) and (26), respectively.

As PVPF allows arbitrary pole placement, the values for ϕ_1 , ϕ_2 , and ω_d can be substituted giving a characteristic equation of the form

$$s^5 + K_4s^4 + K_3s^3 + K_2s^2 + K_1s + K_0. \quad (35)$$

Equating this with equation (32) gives the controller parameters as

$$\begin{aligned} 2\gamma\omega_c &= K_4 - 2\zeta\omega_n \\ \omega_c^2 &= K_3 - \omega_n^2 - 2\zeta\omega_n 2\gamma\omega_c \\ k_t &= K_0/(\sigma^2\omega_c^2) \\ \Gamma_2 &= -(K_2 - 2\zeta\omega_n\omega_c^2 - 2\gamma\omega_c\omega_n^2 - k_t\sigma^2)/\sigma^2 \\ \Gamma_1 &= -(K_1 - \omega_c^2\omega_n^2 - 2\gamma\omega_c k_t\sigma^2)/\sigma^2 \end{aligned} \quad (36)$$

In the following sections, the proposed methods for designing the IRC, PPF, and PVPF control schemes are applied to a nan positioning stage. IFF will not be considered due to the similarity with IRC. The performance of the Butterworth-based controllers is evaluated against the controllers derived using the traditional design methods.

2.5. Note on Parameter Selection

The design methods presented in this section are generalized and contain some undefined parameters. This following will clarify the available choices.

In the case of IRC and IFF, in order to obtain a closed-loop Butterworth filter pattern, the desired closed-loop damping ratios must be defined as $\gamma_1 = 0.9239$ and $\gamma_2 = 0.3827$. This is necessary to ensure a fourth-order closed-loop Butterworth filter pattern. This consequently defines α and β and allows controller parameters to be derived for a given plant.

For PPF and PVPF, the desired closed-loop damping ratios must be defined as: $\phi_1 = 0.8090$ and $\phi_2 = 0.3090$, in order to ensure a fifth-order Butterworth filter pattern is achieved. In the case of PPF, the natural frequency at which the Butterworth filter pattern can be achieved has, thus far, only been found by numerical analysis for a given plant. For PVPF, a closed-loop Butterworth filter pattern can, theoretically, be achieved for any choice of cutoff frequency, ω_d . In practice, it is noted that large choices of ω_d can cause instability. It is recommended that the chosen natural frequency of the Butterworth filter does not exceed the natural frequency of the plant.

3. EXPERIMENTAL SETUP

The performance of the Butterworth-based controller is evaluated on a two-axis serial kinematic nanopositioner, pictured in **Figure 3**. The nanopositioner was designed and constructed at the EasyLab, University of Nevada, Reno. The stage is driven by two 10-mm 200-V piezoelectric stack actuators that provide a range of 40 μm in each axis. The position is measured by a Microsense 6810 capacitive sensor and 6504-01 probe with a sensitivity of 5 $\mu\text{m}/\text{V}$. The stage is driven by two PiezoDrive PDL200 voltage amplifiers with a gain of 20. The control algorithm is implemented using NI LabVIEW's Real-Time module at a sampling rate of 20 kHz.

4. SIMULATIONS

Simulations are performed using a second-order model derived from the measured frequency response, see **Figure 4**, of one-axis of the nanopositioner described in the previous section. The model is calculated to model the dominant resonant mode of the nanopositioner and is given by

$$G(s) = \frac{9.911 \times 10^6}{s^2 + 92.12s + 2.026 \times 10^7}, \quad (37)$$

that is, in terms of the plant parameters of equation (1), $\sigma = 3148.2$, $\zeta = 0.0102$, and $\omega_n = 4501.1$.

The remainder of this section presents the design process of both the traditional and Butterworth-based control schemes for IRC, PPF, and PVPF. IFF has not been included due to the similarity to IRC. The simulations focus on frequency-domain results to ensure stability, and that the closed-loop Butterworth pattern is achieved. Bandwidth measurements are provided for comparison of the traditional and Butterworth-based controllers' performance.

4.1. IRC

4.1.1. Traditional Design

The traditional IRC scheme is designed using the method developed by (Namavar et al., 2014) to achieve maximum damping. The feedthrough term, d , and damping controller gain, are derived as functions of plant parameters. The tracking controller gain is found via trial-and-error such that the closed-loop magnitude response does not exceed unity gain.

$$d = -0.9782 \quad (38)$$

$$C_d(s) = \frac{5472}{s} \quad (39)$$

$$C_t(s) = \frac{950}{s}. \quad (40)$$

4.1.2. Butterworth-Based Design

To achieve a fourth-order Butterworth pattern, the required closed-loop damping ratios are given as

$$\gamma_1 = \cos(\pm 22.5^\circ) = 0.9239 \quad (41)$$

$$\gamma_2 = \cos(\pm 67.5^\circ) = 0.3827. \quad (42)$$

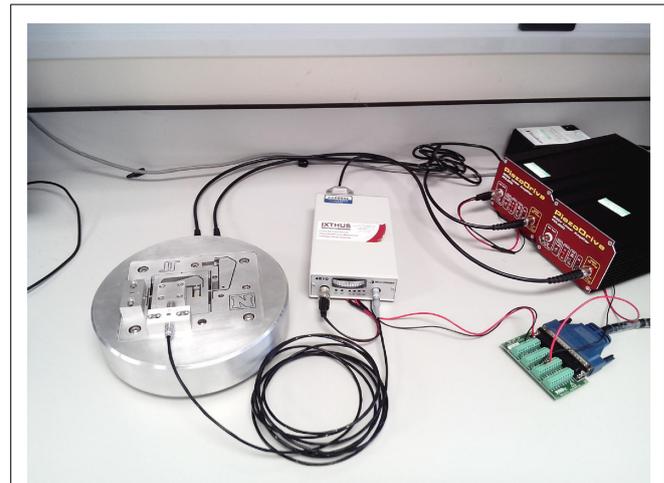


FIGURE 3 | A two-axis serial kinematic nanopositioner, designed at the EasyLab, University of Nevada, Reno, driven by two PiezoDrive 200V Linear amplifiers, with position measured by a Microsense 4810 capacitive sensor.

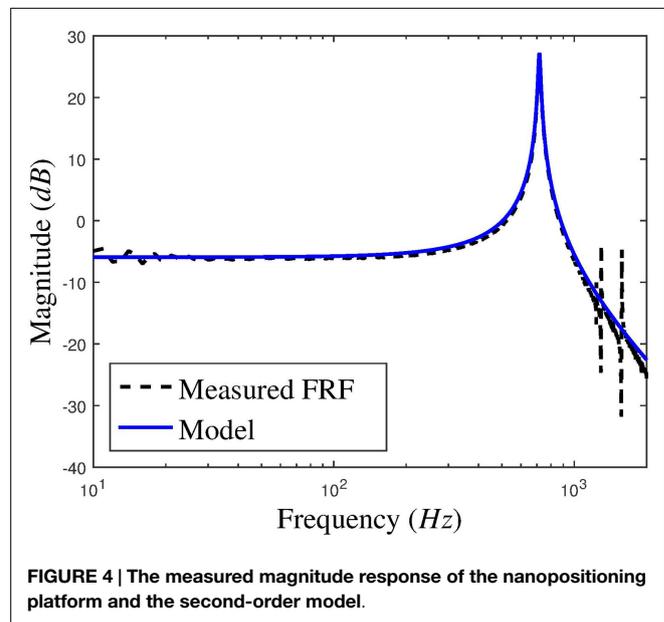


FIGURE 4 | The measured magnitude response of the nanopositioning platform and the second-order model.

From equation (17), the maximum achievable damped natural frequency is found to be $\omega_d = 2471.2$ rad/s. Equations (15), (18), and (19) give the following controllers

$$d = -0.7045 \quad (43)$$

$$C_d(s) = \frac{9035}{s} \quad (44)$$

$$C_t(s) = \frac{416.4}{s}. \quad (45)$$

4.2. PPF

4.2.1. Traditional Design

The traditional PPF scheme is designed using the method found in (Aphale et al., 2008a). In this case, the design criteria are to

maintain the damped natural frequency and increase the damping ratio to 0.37. This damping ratio is chosen as a rough approximation of the achieved damping ratios found in the literature. Damping ratio is used as opposed to coordinates as it is irrespective of system resonances. The tracking controller gain is found via trial-and-error, such that the magnitude response does not exceed 0 dB at any point. This gives the following controllers

$$C_d(s) = \frac{2.403 \times 10^7}{s^2 + 7108s + 3.924 \times 10^7} \quad (46)$$

$$C_t(s) = \frac{1050}{s}. \quad (47)$$

4.2.2. Butterworth-Based Design

The closed-loop damping ratios are

$$\phi_1 = \cos(\pm 36^\circ) = 0.8090 \quad (48)$$

$$\phi_2 = \cos(\pm 72^\circ) = 0.3090. \quad (49)$$

The damped natural frequency, ω_d , is found via simulation. Over a range of ω_d , the desired characteristic equation is calculated and damping and tracking controllers designed using equation (28). The closed-loop transfer function is derived and the poles are found. We choose ω_d as the frequency at which

$$\omega_d - \frac{\sum_{i=1}^5 |s_i|}{5} = 0. \quad (50)$$

This gives $\omega_d = 3655.3$. Equation (28) gives the PPF and tracking controllers as

$$C_d(s) = \frac{5.7 \times 10^7}{s^2 + 1.174 \times 10^4 s + 4.861 \times 10^7} \quad (51)$$

$$C_t(s) = \frac{851}{s}. \quad (52)$$

4.3. PVPF

4.3.1. Traditional Design

The traditional PVPF controller is designed in the same way as the PPF controller, the only difference being the additional parameter

in the numerator relating to velocity. The damping controller is designed to give a closed-loop damping ratio of 0.37, and the tracking controller gain found via trial-and-error to ensure the magnitude response does not exceed 0 dB. The damping and tracking controllers are calculated as follows

$$C_d(s) = \frac{-2248s + 2.403 \times 10^7}{s^2 + 7108s + 3.924 \times 10^7} \quad (53)$$

$$C_t(s) = \frac{1500}{s}. \quad (54)$$

4.3.2. Butterworth-Based Design

The closed-loop damping ratios are

$$\phi_1 = \cos(\pm 36^\circ) = 0.8090 \quad (55)$$

$$\phi_2 = \cos(\pm 72^\circ) = 0.3090, \quad (56)$$

and the damped natural frequency is set to $\omega_d = \omega_n$. This gives the following characteristic equation

$$s^5 + 1.4567 \times 10^4 s^4 + 1.061 \times 10^8 s^3 + 4.776 \times 10^{11} s^2 + 1.3287 \times 10^{15} s + 1.8482 \times 10^{18}. \quad (57)$$

The controllers are derived from equation (36) as

$$C_d(s) = \frac{-1.56 \times 10^4 s + 7.065 \times 10^7}{s^2 + 1.447 \times 10^4 s + 8.45 \times 10^7} \quad (58)$$

$$C_t(s) = \frac{2207}{s} \quad (59)$$

The simulated closed-loop frequency response of both the traditional and Butterworth-based control schemes of IRC, PPF, and PVPF are presented in **Figure 5** and the measured bandwidth given in **Table 1**. The bandwidth is defined as the lowest frequency at which the magnitude response of the system breaches the specified limits. Both positive and negative limits are imposed as an increase in magnitude can cause performance degradation equal to that of a reduction in magnitude. For nanopositioning applications, which require highly accurate tracking, the standard -3 dB bandwidth is not sufficiently stringent (Namavar et al., 2014). For this reason, the ± 1 dB bandwidth is considered as a more pertinent performance metric. The ± 3 dB bandwidth is also included for comparison. **Figure 6** shows the closed-loop pole-zero map of the Butterworth-based designs.

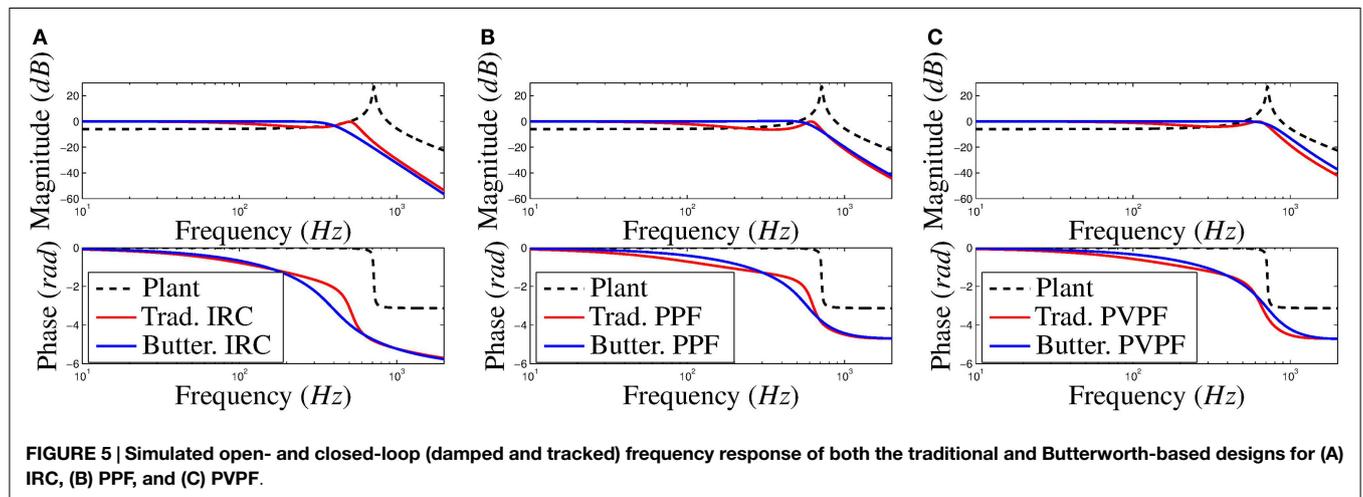


FIGURE 5 | Simulated open- and closed-loop (damped and tracked) frequency response of both the traditional and Butterworth-based designs for (A) IRC, (B) PPF, and (C) PVPF.

TABLE 1 | Simulation results.

	IRC		PPF		PVPF	
	Trad.	Butter.	Trad.	Butter.	Trad.	Butter.
Bandwidth ± 1 dB (Hz)	76	332	64	552	92	658
Bandwidth ± 3 dB (Hz)	164	392	130	612	204	738

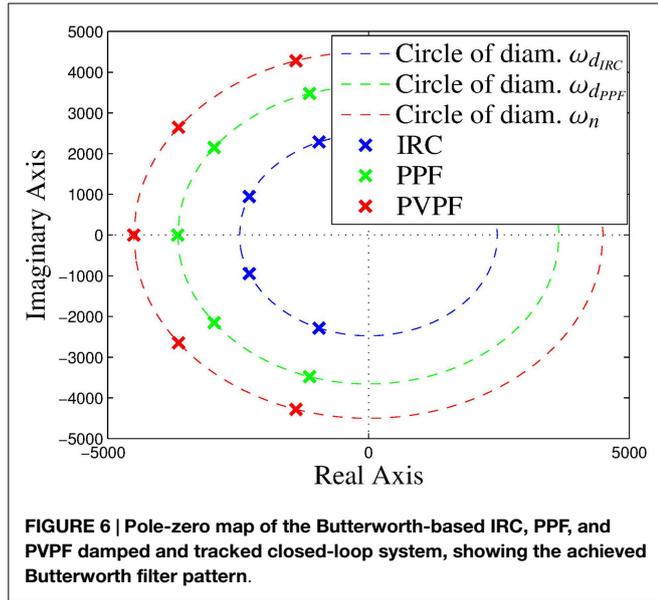


FIGURE 6 | Pole-zero map of the Butterworth-based IRC, PPF, and PVPF damped and tracked closed-loop system, showing the achieved Butterworth filter pattern.

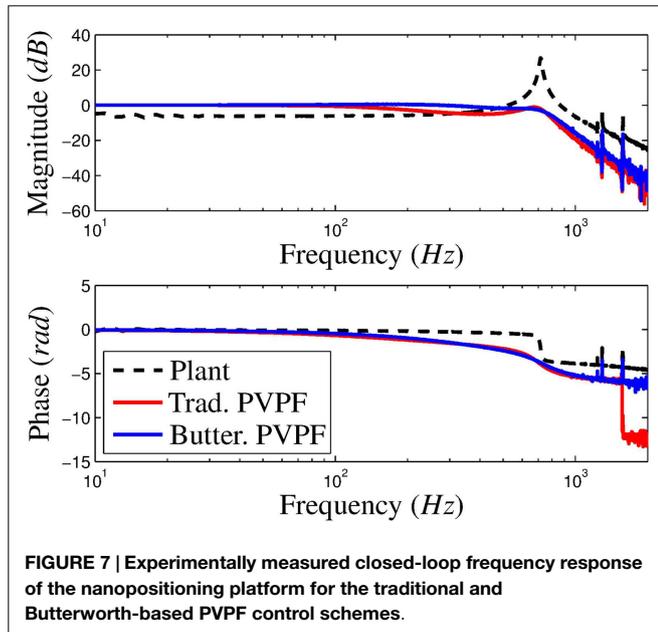


FIGURE 7 | Experimentally measured closed-loop frequency response of the nanopositioning platform for the traditional and Butterworth-based PVPF control schemes.

5. EXPERIMENTAL RESULTS

Experiments were performed using the nanopositioning platform as described in Section 3. The performance of the control scheme was evaluated in both the frequency- and time-domain. In Section 2, it was found that in the case of IRC and PPF, the

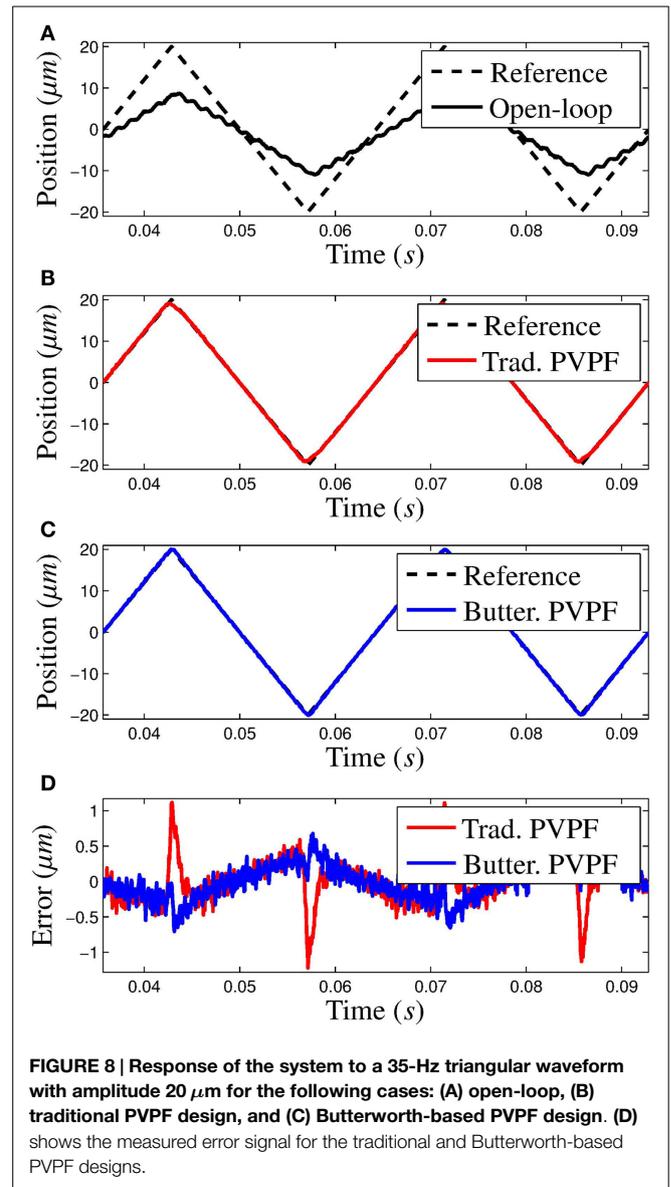


FIGURE 8 | Response of the system to a 35-Hz triangular waveform with amplitude $20 \mu\text{m}$ for the following cases: (A) open-loop, (B) traditional PVPF design, and (C) Butterworth-based PVPF design. (D) shows the measured error signal for the traditional and Butterworth-based PVPF designs.

closed-loop Butterworth filter pattern can only be achieved for a single cutoff frequency, ω_d . The application in Section 4 shows that the cutoff frequency attained using IRC and PPF is significantly less than that of the PVPF scheme. Additionally, using PVPF a closed-loop Butterworth filter pattern can be achieved for a large range of natural frequencies. For this reason, the PVPF based control scheme is deemed to be of greater use, and, as such, is the sole consideration of experimental analysis.

5.1. Frequency-Domain Results

The closed-loop frequency response function of both the traditional and Butterworth-based PVPF control schemes is plotted in Figure 7. The implementation is seen to be stable. It is observed that the results largely match the simulated results. The Butterworth-based PVPF control scheme does not display the frequency response of a perfect Butterworth filter.

TABLE 2 | Experimental results.

	Trad.	Butter.
Bandwidth ± 1 dB (Hz)	123	370
Bandwidth ± 3 dB (Hz)	215	720
Max. error (μm)	1.1161	0.6777
RMS error (μm)	0.2892	0.2436

There is a minor increase in magnitude from approximately 100 to 250 Hz, followed by a premature decrease from approximately 300 to 700 Hz. Thorough testing of the experimental platform indicates that this can be attributed to the delay found in the controller implementation. This accounts for the relatively low ± 1 dB bandwidth although the ± 3 dB bandwidth is sufficiently similar to that predicted by simulation. In comparison with the traditionally designed PVPF control scheme, it is observed that the achievable bandwidth is significantly increased.

5.2. Time-Domain Results

To measure the time-domain performance of the respective control schemes, the platform is subjected to a triangular input waveform, as is typical in nanositioning applications. The triangular wave has a frequency of 35 Hz and an amplitude of 20 μm . The phase-corrected system response and error are shown in **Figure 8** and the respective maximum and RMS error measurements are provided in **Table 2**.

The open-loop response, plotted in **Figure 8A**, shows the triangular wave excites the resonance of the plant. The gain of the plant is significantly less than unity, and so, provides no tracking. Comparing the response of the traditional PVPF, **Figure 8B**, and the Butterworth-based PVPF, **Figure 8C**, it is observed that the Butterworth-based control scheme tracks more accurately, especially at the turn-around point. This is reflected in the error signal (**Figure 8D**) and the maximum recorded error (see **Table 2**). The traditional PVPF scheme

shows a significant deviation from the reference, whereas the error of the Butterworth-based scheme at the turn-around is nearly indistinguishable from the rest of the scan. This effect becomes more pronounced at higher scan frequencies, due to the dip in the magnitude response of the traditional PVPF scheme. This causes a reduction of the gain in the lower harmonics of the triangle waveform. The relatively flat response of the Butterworth-based control scheme ensures the harmonics of the triangle wave will be preserved as the scanning speed is increased.

6. CONCLUSION

In this paper, a method is derived to obtain a Butterworth filter pattern in the damped and tracked response of a second-order system for each of the following four control schemes: Integral Resonant Control, Integral Force Feedback, Positive Position Feedback, and Positive Velocity and Position Feedback. Simulations show that the desired pole placement is achieved and this is reflected in frequency response. Using this method, a higher bandwidth can be achieved, which allows a greater accuracy in positioning. This is confirmed by experimental results. Due to the simplicity of the method, this is ideal for improving the performance of precise positioning systems.

AUTHOR CONTRIBUTIONS

DR developed the theory of Butterworth-based simultaneous design of damping and tracking controllers presented in the paper and drafted the submitted manuscript. AS-MR conducted all of the experimental work and helped with the development of the theory and writing. SA supervised DR for theoretical development as well as AS-MR for the experimental setup and validation. VF provided guidance regarding theoretical development and gave inputs into paper writing.

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Conflict of Interest Statement: The authors declare that the research was conducted in the absence of any commercial or financial relationships that could be construed as a potential conflict of interest.

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